

## Boundary value problems

**Example 127.** The IVP (initial value problem)  $y'' + 4y = 0$ ,  $y(0) = 0$ ,  $y'(0) = 0$  has the unique solution  $y(x) = 0$ .

Initial value problems are often used when the problem depends on time. Then,  $y(0)$  and  $y'(0)$  describe the initial configuration at  $t = 0$ .

For problems which instead depend on spatial variables, such as position, it may be natural to specify values at positions on the boundary (for instance, if  $y(x)$  describes the steady-state temperature of a rod at position  $x$ , we might know the temperature at the two end points).

The next example illustrates that such a boundary value problem (BVP) may or may not have a unique solution.

**Example 128.** Verify the following claims.

- (a) The BVP  $y'' + 4y = 0$ ,  $y(0) = 0$ ,  $y(1) = 0$  has the unique solution  $y(x) = 0$ .
- (b) The BVP  $y'' + \pi^2 y = 0$ ,  $y(0) = 0$ ,  $y(1) = 0$  is solved by  $y(x) = B \sin(\pi x)$  for any value  $B$ .

**Solution.**

- (a) We know that the general solution to the DE is  $y(x) = A \cos(2x) + B \sin(2x)$ . The boundary conditions imply  $y(0) = A \stackrel{!}{=} 0$  and, using that  $A = 0$ ,  $y(1) = B \sin(2) \stackrel{!}{=} 0$  shows that  $B = 0$  as well.
- (b) This time, the general solution to the DE is  $y(x) = A \cos(\pi x) + B \sin(\pi x)$ . The boundary conditions imply  $y(0) = A \stackrel{!}{=} 0$  and, using that  $A = 0$ ,  $y(1) = B \sin(\pi) \stackrel{!}{=} 0$ . This second condition is true for every  $B$ .

It is therefore natural to ask: for which  $\lambda$  does the BVP  $y'' + \lambda y = 0$ ,  $y(0) = 0$ ,  $y(L) = 0$  have nonzero solutions? (We assume that  $L > 0$ .)

Such solutions are called **eigenfunctions** and  $\lambda$  is the corresponding **eigenvalue**.

**Remark.** Compare that to our previous use of the term eigenvalue: given a matrix  $A$ , we asked: for which  $\lambda$  does  $Av - \lambda v = 0$  have nonzero solutions  $v$ ? Such solutions were called eigenvectors and  $\lambda$  was the corresponding eigenvalue.

**Example 129.** Find all eigenfunctions and eigenvalues of  $y'' + \lambda y = 0$ ,  $y(0) = 0$ ,  $y(L) = 0$ .

Such a problem is called an **eigenvalue problem**.

**Solution.** The solutions of the DE look different in the cases  $\lambda < 0$ ,  $\lambda = 0$ ,  $\lambda > 0$ , so we consider them individually.

$\lambda = 0$ . Then  $y(x) = Ax + B$  and  $y(0) = y(L) = 0$  implies that  $y(x) = 0$ . No eigenfunction here.

$\lambda < 0$ . The roots of the characteristic polynomial are  $\pm\sqrt{-\lambda}$ . Writing  $\rho = \sqrt{-\lambda}$ , the general solution therefore is  $y(x) = Ae^{\rho x} + Be^{-\rho x}$ .  $y(0) = A + B \stackrel{!}{=} 0$  implies  $B = -A$ . Using that, we get  $y(L) = A(e^{\rho L} - e^{-\rho L}) \stackrel{!}{=} 0$ . For eigenfunctions we need  $A \neq 0$ , so  $e^{\rho L} = e^{-\rho L}$  which implies  $\rho L = -\rho L$ . This cannot happen since  $\rho \neq 0$  and  $L \neq 0$ . Again, no eigenfunctions in this case.

$\lambda > 0$ . The roots of the characteristic polynomial are  $\pm i\sqrt{\lambda}$ . Writing  $\rho = \sqrt{\lambda}$ , the general solution thus is  $y(x) = A \cos(\rho x) + B \sin(\rho x)$ .  $y(0) = A \stackrel{!}{=} 0$ . Using that,  $y(L) = B \sin(\rho L) \stackrel{!}{=} 0$ . Since  $B \neq 0$  for eigenfunctions, we need  $\sin(\rho L) = 0$ . This happens if  $\rho L = n\pi$  for  $n = 1, 2, 3, \dots$  (since  $\rho$  and  $L$  are both positive,  $n$  must be positive as well). Equivalently,  $\rho = \frac{n\pi}{L}$ . Consequently, we find the eigenfunctions  $y_n(x) = \sin\left(\frac{n\pi x}{L}\right)$ ,  $n = 1, 2, 3, \dots$ , with eigenvalue  $\lambda = \left(\frac{n\pi}{L}\right)^2$ .

**Example 130.** Suppose that a rod of length  $L$  is compressed by a force  $P$  (with ends being pinned [not clamped] down). We model the shape of the rod by a function  $y(x)$  on some interval  $[0, L]$ . The theory of elasticity predicts that, under certain simplifying assumptions,  $y$  should satisfy  $EIy'' + Py = 0$ ,  $y(0) = 0$ ,  $y(L) = 0$ .

Here,  $EI$  is a constant modeling the inflexibility of the rod ( $E$ , known as Young's modulus, depends on the material, and  $I$  depends on the shape of cross-sections (it is the area moment of inertia)).

In other words,  $y'' + \lambda y = 0$ ,  $y(0) = 0$ ,  $y(L) = 0$ , with  $\lambda = \frac{P}{EI}$ .

The fact that there is no nonzero solution unless  $\lambda = \left(\frac{\pi n}{L}\right)^2$  for some  $n = 1, 2, 3, \dots$ , means that buckling can only occur if  $P = \left(\frac{\pi n}{L}\right)^2 EI$ . In particular, no buckling occurs for forces less than  $\frac{\pi^2 EI}{L^2}$ . This is known as the critical load (or Euler load) of the rod.

**Comment.** This is a very simplified model. In particular, it assumes that the deflections are small. (Technically, the buckled rod in our model is longer than  $L$ ; of course, that's not the case in practice.)

[https://en.wikipedia.org/wiki/Euler%27s\\_critical\\_load](https://en.wikipedia.org/wiki/Euler%27s_critical_load)